

Modeliranje turbulencije u cilju primene numeričkih simulacija u hidrotehnici

Univerzitet u Beogradu

Građevinski fakultet

- Kurs Mehanike fluida na doktorskim studijama -

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CFD – Computational Fluid Mechanics

Računska mehanika fluida

Primena metoda numeričke analize za rešavanje jednačina održanja:

- Jednačina kontinuiteta
- Dinamička jednačina
- Jednačine transporta

Zašto je potrebno „modeliranje“ turbulencije ?

Primer: Modeliranje dvofaznog strujanja voda/vazduh:

- **Mass conservation equations:**

$$\frac{\partial \alpha_w}{\partial t} + \frac{\partial(\alpha_w V_{wi})}{\partial x_i} = \frac{G_{gw}}{\rho_w}$$

$$\frac{\partial(\rho_g \alpha_g)}{\partial t} + \frac{\partial(\rho_g \alpha_g V_{gj})}{\partial x_j} = -G_{gw}$$

$$\alpha_w + \alpha_g = 1 \quad \rho_g = \frac{p_g M_g}{RT}$$

α_w - water phase content

α_g - oxygen phase content

G_{gw} - Mass transfer of oxygen from the gas to the water phase

- **Momentum equations:**

- **water phase:**

$$\alpha_w \frac{\partial V_{wi}}{\partial t} + \alpha_w V_{wj} \frac{\partial V_{wi}}{\partial x_j} = -\frac{\alpha_w}{\rho_w} \frac{\partial p_w}{\partial x_i} + \alpha_w g_i + \frac{1}{\rho_w} \frac{\partial(\alpha_w \tau_{ij})}{\partial x_j} + \frac{1}{\rho_w} F_{gw}$$

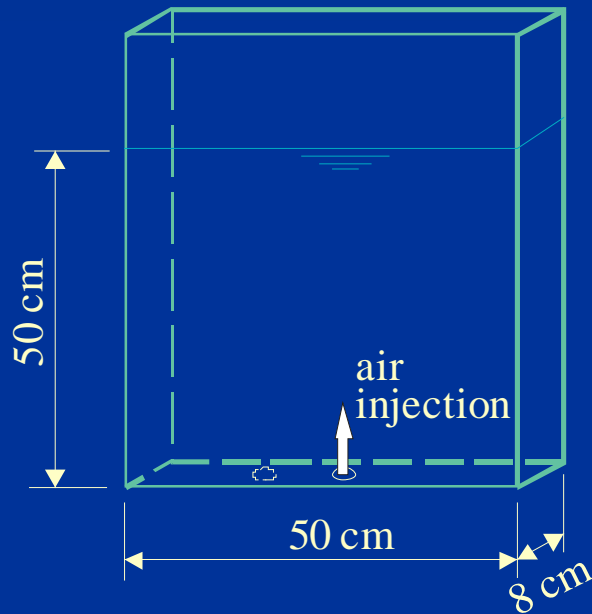
- **gas phase:**

$$\frac{\partial(\alpha_g \rho_g V_{gi})}{\partial t} + \frac{\partial(\alpha_g \rho_g V_{gi} V_{gj})}{\partial x_j} = -\alpha_g \frac{\partial p_g}{\partial x_i} + \alpha_g \rho_g g_i - F_{gw}$$

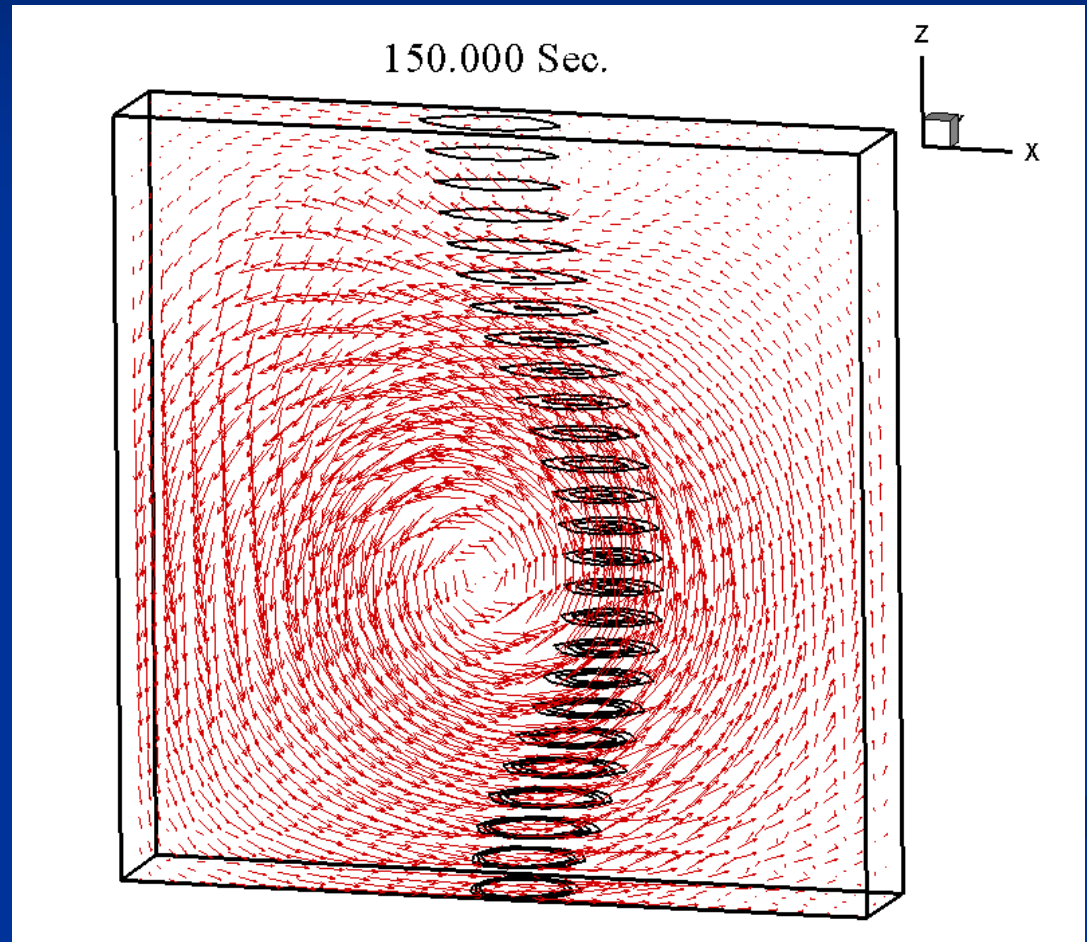
Momentum exchange



Modeliranje dvofaznog strujanja voda/vazduh:

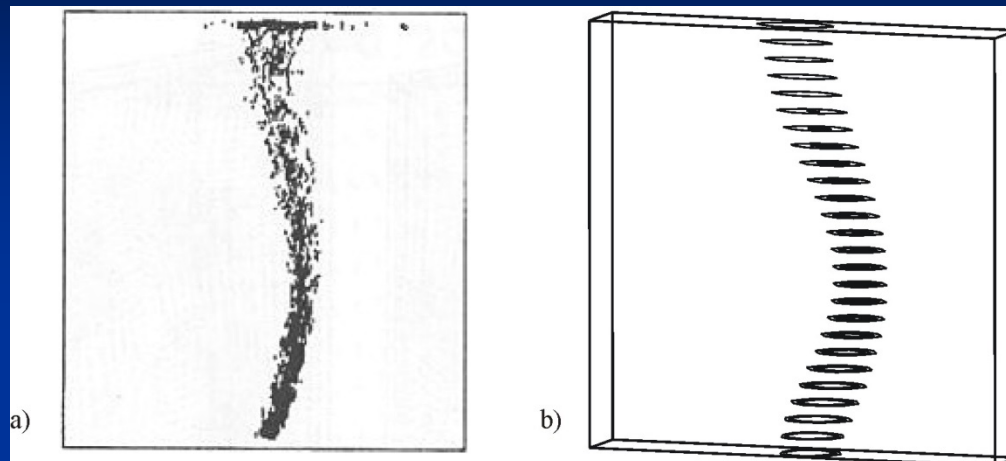


$$Q_{\text{air}} = 1.0 \text{ L/min}$$

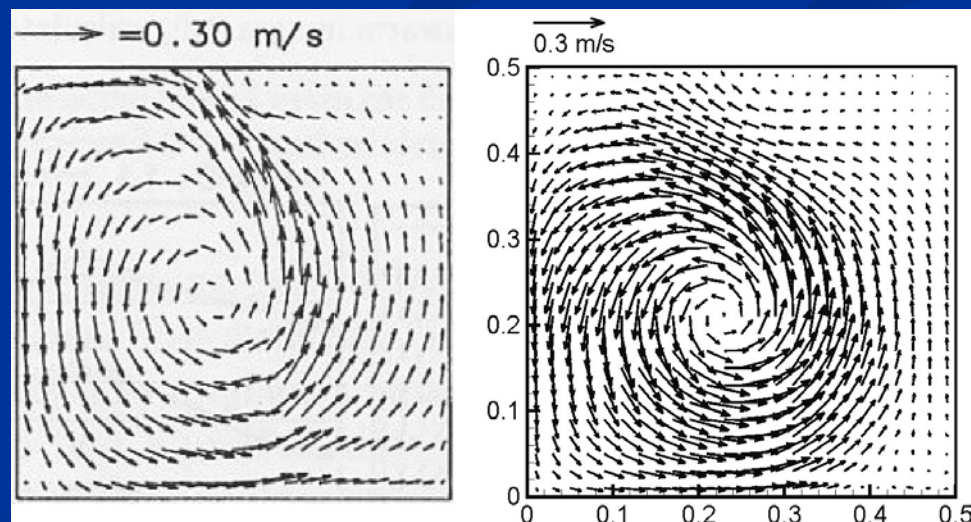


Poređenje sa merenjima:

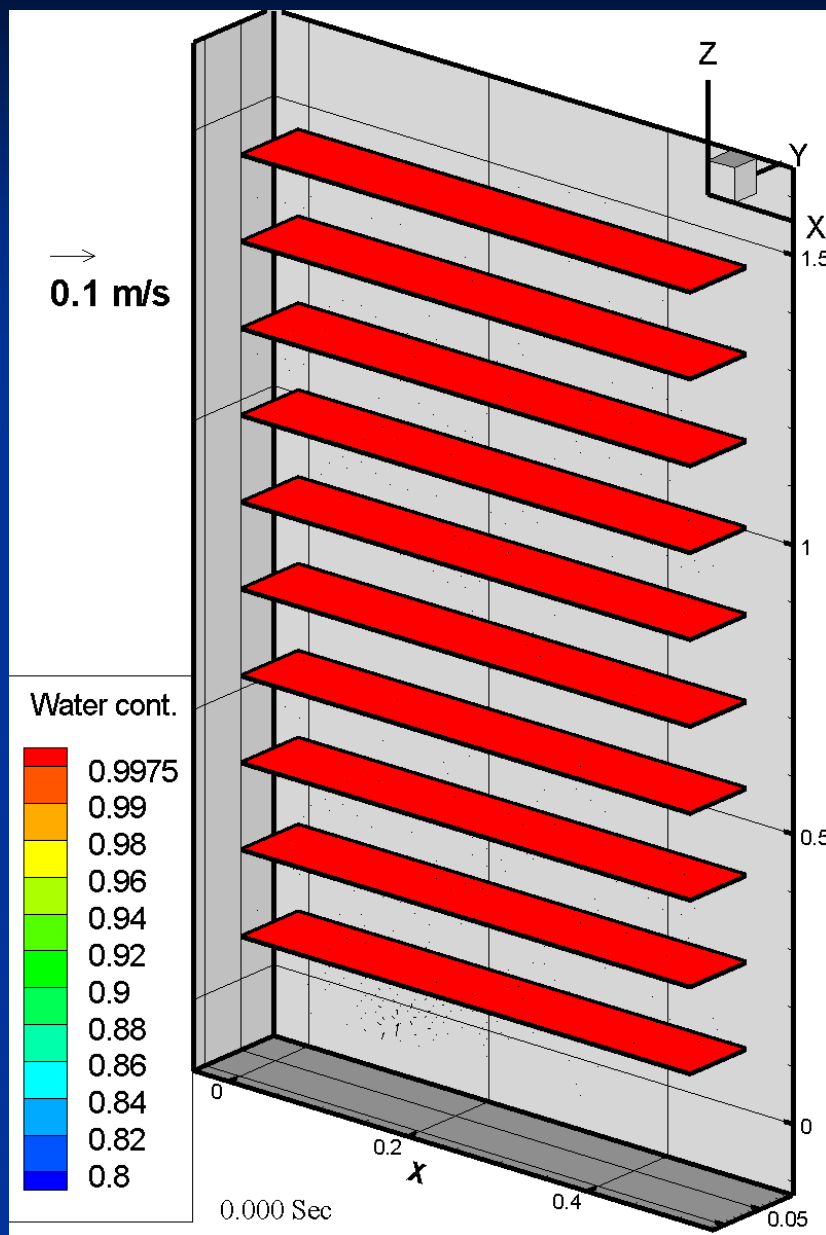
Poređenje oblika
vazdušne struje



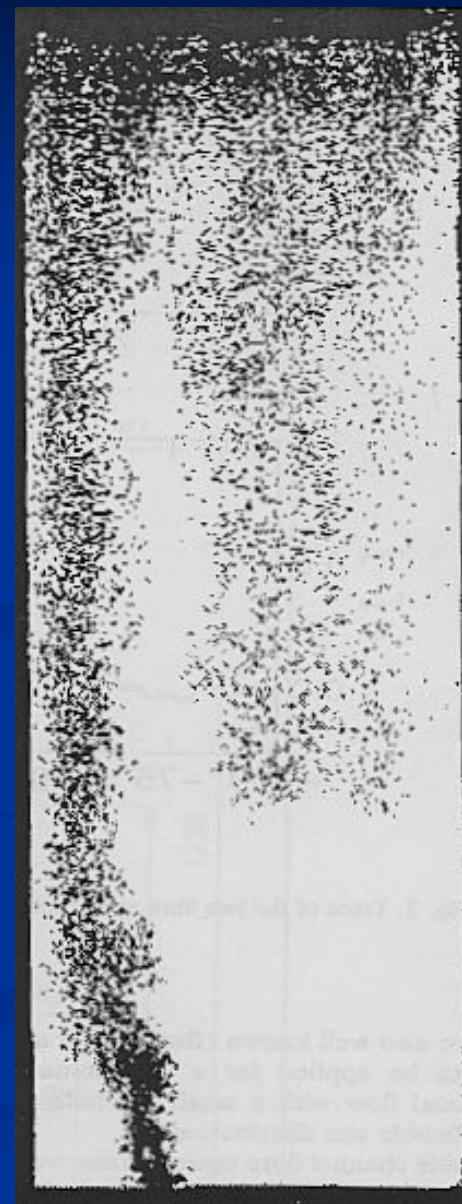
Poređenje strujne slike
tečne faze



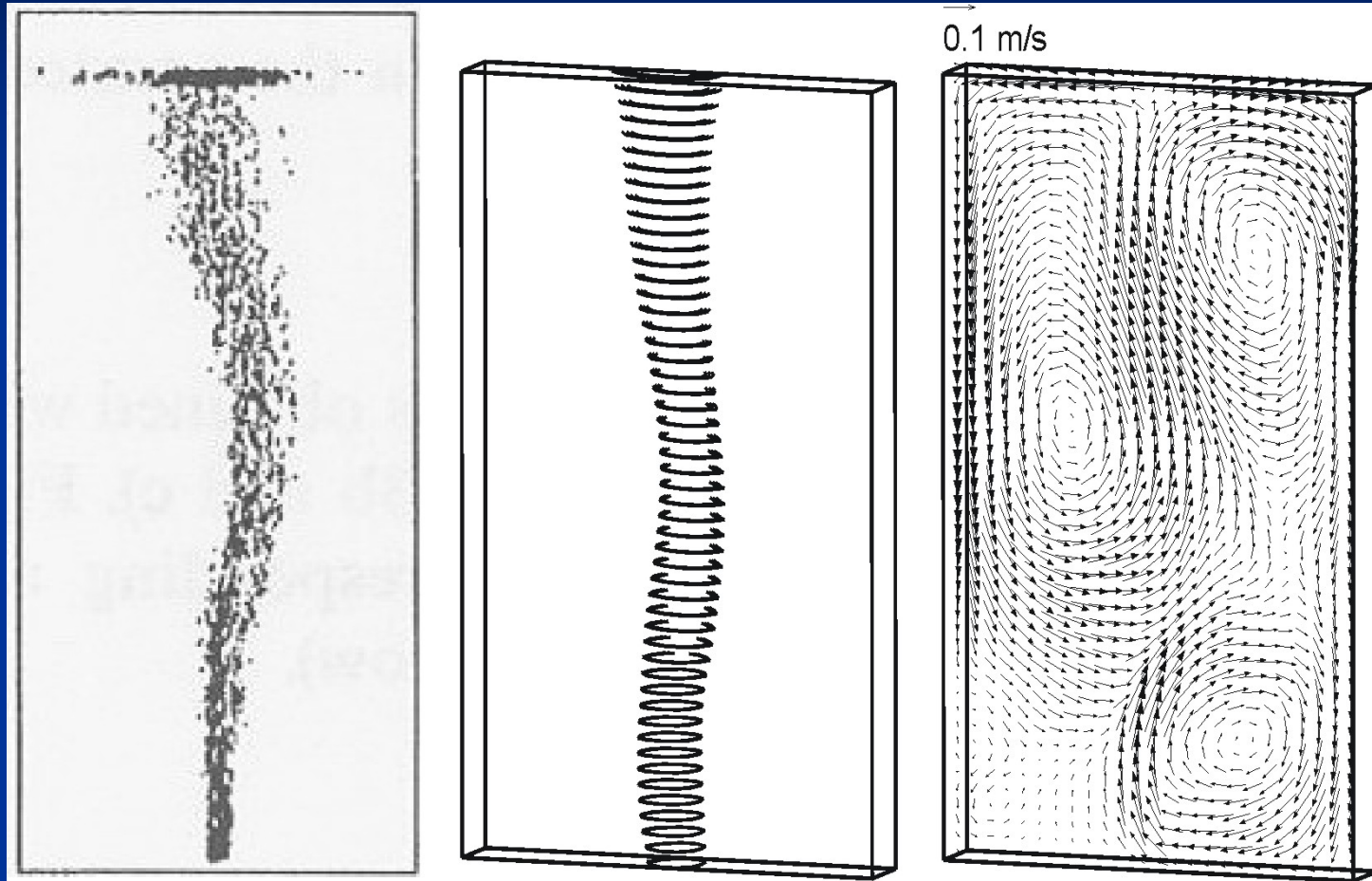
3D simulacija eksperimenta:



Osmotreni raspored mehurića vazduha:



Poređenje rezultata simulacije sa merenjima



$Q_{\text{air}} = 1 \text{ L/min}$

Rejnoldsove jednačine:

$$\rho \frac{\partial U_i}{\partial t} + \rho \frac{\partial (U_i U_j)}{\partial x_j} = - \frac{\partial p_w}{\partial x_i} + \rho g_i + \mu \frac{\partial^2 U_i}{\partial x_j \partial x_j} - \frac{\partial}{\partial x_j} \overline{\rho u_i' u_j'}$$

$$\frac{\partial U_i}{\partial x_i} = 0 \quad u_i = U_i + u_i'$$

- Modeliranje Rejnoldsovih napona predstavlja ključni element zatvaranja sistema jednačina, koje se potom rešavaju metodama numeričke analize.
- Dva su osnovna efekta turbulencije na glavni tok:
 - Oduzima energiju glavnog toka;
 - Doprinosi transportu mase, količine kretanja ili energije upravno na glavni tok.

Dakle, efekti su isti kao i u slučaju molekularne viskoznosti (npr. kod laminarnog strujanja).

- Na osnovu toga, prirodno je pretpostaviti da se efekti turbulencije na glavni tok, predstavljeni Rejnoldsovim naponima u osrednjenim jednačinama, mogu „modelirati“ analogno viskoznim naponima (koncept turbulentne viskoznosti – Boussinesq/1877.).

$$-\overline{u_i' u_j'} = \nu_T \left(\frac{\partial U_i}{\partial x_j} + \frac{\partial U_j}{\partial x_i} \right) - \frac{2}{3} \delta_{ij} k \quad k = \frac{1}{2} \overline{u_i' u_i'}$$

- Na osnovu dimenzionalne analize može se zaključiti:

$$\nu_T = C \nu_T L_T$$

- Na ovaj način se problem modeliranja turbulencije sveo na problem ocene karakteristične brzine i dužine turbulencije u svakoj tački toka.

Na sličan način se može modelirati i transport rastvorene materije usled turbulentnih fluktuacija

➤ Osrednjena jednačina „transporta“

$$\frac{\partial \Phi}{\partial t} + \frac{\partial (\Phi U_j)}{\partial x_j} = \frac{\partial}{\partial x_j} \left(D \frac{\partial \Phi}{\partial x_j} \right) + \frac{\partial}{\partial x_j} \left(-\overline{u'_j \phi'} \right)$$

$$\phi = \Phi + \phi'$$

➤ Turbulentna difuzija se može modelirati analogno molekularnoj difuziji:

$$-\overline{u'_j \phi'} = D_T \frac{\partial \Phi}{\partial x_j}$$

gde se D_T obično daje u funkciji turbulentne viskoznosti:

$$D_T = \frac{V_T}{\sigma}$$

Malo istorije

- Prandtl (1925) – koncept dužine mešanja (algebarski izraz za Reynoldsov napon);
- Prandtl (1945) – modeliranje turbulencije rešavanjem jedne dodatne dif. jednačine koja opisuje transport i promenu kinetičke energije turbulencije u toku;
- Kolmogorov (1942) – modeliranje turbulencije rešavanjem dve dodatne dif. jednačine koje opisuju transport i promenu kinetičke energije turbulencije i disipacije energije u toku (k- ω model);

$$L_T \approx \frac{k^{1/2}}{\omega}$$

k-ε model

(Harlow & Nakayama, 1968)

(Launder & Spalding, 1972)

➤ Jednačina kinetičke energije fluktuacija (k jednačina)

$$\rho \frac{\partial k}{\partial t} + \rho \frac{\partial (kU_j)}{\partial x_j} = -\overline{\rho u_i' u_j'} \frac{\partial U_i}{\partial x_j} - \underbrace{\mu \frac{\partial u_i'}{\partial x_k} \frac{\partial u_i'}{\partial x_k}}_{\text{disipacija k}} + \frac{\partial}{\partial x_j} \left(\underbrace{\mu \frac{\partial k}{\partial x_j}}_{\text{difuzija k}} - \frac{1}{2} \overline{\rho u_i' u_i' u_j'} - \overline{p' u_j'} \right)_{\text{transport k usled fluktuacija brzine i pritiska}}$$

lokalna i konv.
promena k

disipacija k
($\rho\varepsilon$)

transport k
usled
fluktuacija
brzine i
pritiska

produkcija k

difuzija k

$$-\overline{\rho u_i' u_j'} = \mu_T \left(\frac{\partial U_i}{\partial x_j} + \frac{\partial U_j}{\partial x_i} \right) - \frac{2}{3} \rho \delta_{ij} k$$

$$\mu_T = C_\mu \rho k^{1/2} L_T \quad \varepsilon = C_D \frac{k^{3/2}}{L_T}$$

$$\frac{\mu_T}{\sigma_k} \frac{\partial k}{\partial x_j} = \left(-\frac{1}{2} \overline{\rho u_i' u_i' u_j'} - \overline{p' u_j'} \right)$$

ε jednačina

produkcija ε (P_ε)

$$\rho \frac{\partial \varepsilon}{\partial t} + \rho \frac{\partial(\varepsilon U_j)}{\partial x_j} = -2\mu \left[\overline{\frac{\partial u'_i}{\partial x_k} \frac{\partial u'_i}{\partial x_k}} + \overline{\frac{\partial u'_k}{\partial x_i} \frac{\partial u'_k}{\partial x_i}} \right] \frac{\partial U_i}{\partial x_j} - 2\mu \overline{u'_k} \frac{\partial u'_i}{\partial x_j} \frac{\partial^2 U_i}{\partial x_k \partial x_j} -$$

lokalna i konv.
promena ε

$$-2\mu \overline{\frac{\partial u'_i}{\partial x_j} \frac{\partial u'_i}{\partial x_k} \frac{\partial u'_j}{\partial x_k}} - 2\mu \nu \overline{\frac{\partial^2 u'_i}{\partial x_j \partial x_k} \frac{\partial^2 u'_i}{\partial x_j \partial x_k}} +$$

disipacija ε

$$+ \frac{\partial}{\partial x_j} \left(\mu \frac{\partial \varepsilon}{\partial x_j} - \overline{\mu u'_j \frac{\partial u'_i}{\partial x_k} \frac{\partial u'_i}{\partial x_k}} - 2\nu \overline{\frac{\partial p'}{\partial x_k} \frac{\partial u'_j}{\partial x_k}} \right)$$

difuzija ε

transport ε usled
fluktuacija brzine i
pritiska

Uvodi se pretpostavka o
lokalnoj ravnoteži :

$$P_k \approx \varepsilon$$

$$P_\varepsilon \approx \frac{P_k}{T_T}$$

$$T_T \approx \frac{k}{\varepsilon}$$

produkcija ε (P_ε)

$$P_\varepsilon = -2\mu \left[\overline{\frac{\partial u'_i}{\partial x_k} \frac{\partial u'_i}{\partial x_k}} + \overline{\frac{\partial u'_k}{\partial x_i} \frac{\partial u'_k}{\partial x_i}} \right] \frac{\partial U_i}{\partial x_j} - 2\mu \overline{u'_k} \frac{\partial u'_i}{\partial x_j} \frac{\partial^2 U_i}{\partial x_k \partial x_j} = C_{\varepsilon 1} \frac{k}{\varepsilon} P_k$$

disipacija ε

$$D_\varepsilon = -2\mu \overline{\frac{\partial u'_i}{\partial x_j} \frac{\partial u'_i}{\partial x_k} \frac{\partial u'_j}{\partial x_k}} - 2\mu \nu \overline{\frac{\partial^2 u'_i}{\partial x_j \partial x_k} \frac{\partial^2 u'_i}{\partial x_j \partial x_k}} \approx \frac{\varepsilon}{T_T} \quad D_\varepsilon = C_{\varepsilon 2} \frac{\varepsilon^2}{k}$$

disperzija ε

$$\frac{\partial}{\partial x_j} \left(\mu \frac{\partial \varepsilon}{\partial x_j} - \mu \overline{u'_j \frac{\partial u'_i}{\partial x_k} \frac{\partial u'_i}{\partial x_k}} - 2\nu \overline{\frac{\partial p'}{\partial x_k} \frac{\partial u'_j}{\partial x_k}} \right) = \frac{\partial}{\partial x_j} \left(\mu \frac{\partial \varepsilon}{\partial x_j} + \frac{\mu_T}{\sigma_\varepsilon} \frac{\partial \varepsilon}{\partial x_j} \right)$$

Rezime (k-ε model)

➤ k jednačina:

$$\rho \frac{\partial k}{\partial t} + \rho \frac{\partial (kU_j)}{\partial x_j} = P_k - \rho \varepsilon + \frac{\partial}{\partial x_j} \left(\mu \frac{\partial k}{\partial x_j} + \frac{\mu_T}{\sigma_k} \frac{\partial k}{\partial x_j} \right)$$

➤ ε jednačina:

$$\rho \frac{\partial \varepsilon}{\partial t} + \rho \frac{\partial (\varepsilon U_j)}{\partial x_j} = C_{\varepsilon 1} \frac{\varepsilon}{k} P_k - C_{\varepsilon 2} \rho \frac{\varepsilon^2}{k} + \frac{\partial}{\partial x_j} \left(\mu \frac{\partial \varepsilon}{\partial x_j} + \frac{\mu_T}{\sigma_\varepsilon} \frac{\partial \varepsilon}{\partial x_j} \right)$$

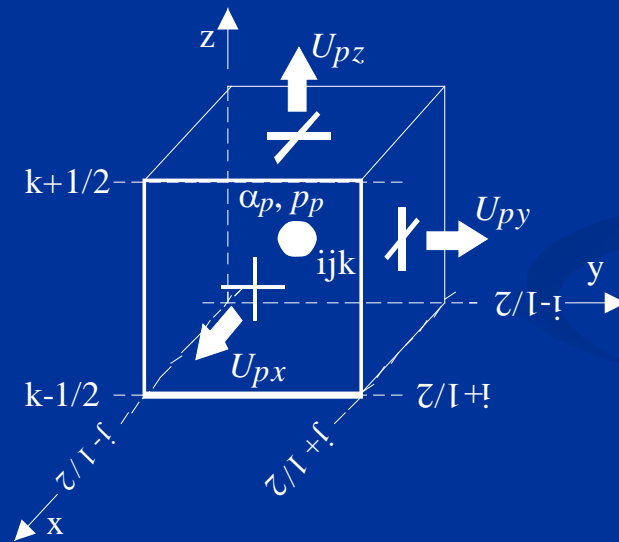
$$\mu_T = C_\mu \rho \frac{k^2}{\varepsilon}$$

$$C_{\varepsilon 1} = 1.44 \quad C_{\varepsilon 2} = 1.92 \quad C_\mu = 0.09 \quad \sigma_k = 1.0 \quad \sigma_\varepsilon = 1.3$$

Sada, kada znamo kako „modelirati“ turbulenciju, preostaje numeričko rešavanje osnovnih jednačina

$$\frac{\partial U_i}{\partial t} + \frac{\partial U_i U_j}{\partial x_j} = -\frac{1}{\rho} \frac{\partial p}{\partial x_i} + g_i + \frac{1}{\rho} \frac{\partial}{\partial x_j} (\tau_{ij}^{\mu} + \tau_{ij}^T)$$

$$\frac{\partial U_i}{\partial x_i} = 0$$



➤ HSMAC shema za proračun nepoznatih brzina i pritisaka u strujnom polju

Iterativna shema se sastoji iz tri koraka:

➤ **1. korak:**

$$\frac{(U_i)^* - (U_i)^n}{\Delta t} = (\text{inertia } t.)^n + (\text{pressure } t.)^{n+1} + (\text{viscous } t.)^n + (\text{turb } t.)^n$$

➤ **2. korak:**

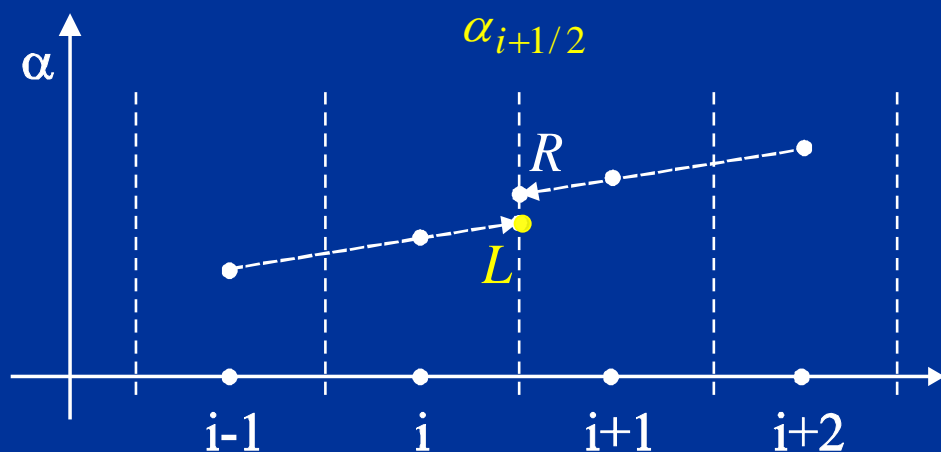
$$D = \frac{\partial U_i^*}{\partial x_i}$$

➤ **3. korak:**

$$p^{n+1,r+1} = p^{n+1,r} - \omega D / \left(\frac{\partial D}{\partial p} \right) = p^{n+1,r} - \frac{\omega D^{r+1}}{2\Delta t \left(\frac{1}{\Delta x_1^2} + \frac{1}{\Delta x_2^2} + \frac{1}{\Delta x_3^2} \right)}$$

Problem konvektivnih članova u jednačinama (ili kako umanjiti numeričku difuziju, a pri tome izbeći oscilacije rešenja)

➤ **Scheme višeg reda tačnosti sa tzv. TVD limiterima:**

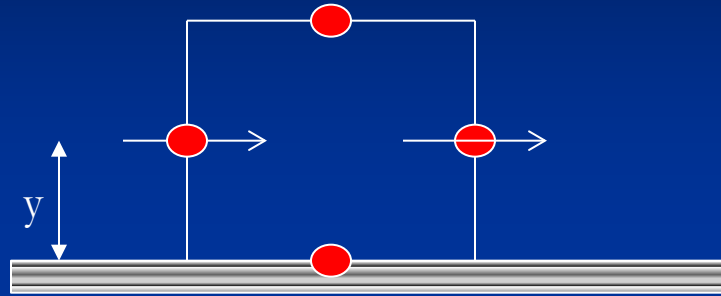


$$\alpha_{i+1/2}^L = \alpha_i + \frac{1}{2}S$$

"Superbee" limiter:

$$S = \max \begin{cases} 0 \\ \min [2(\alpha_{i+1} - \alpha_i), (\alpha_i - \alpha_{i-1})] \\ \min [(\alpha_{i+1} - \alpha_i), 2(\alpha_i - \alpha_{i-1})] \end{cases}$$

Granični uslov uz čvrstu granicu



➤ **gladak zid:**

$$\frac{U(y)}{U_*} = \frac{1}{\kappa} \ln \left(\frac{yU_*}{\nu} \right) + 5.5$$

➤ **hrapav zid:**

$$\frac{U(y)}{U_*} = \frac{1}{\kappa} \ln \left(\frac{y}{k_s} \right) + 8.5$$

$$k = \frac{U_*^2}{\sqrt{C_\mu}}$$

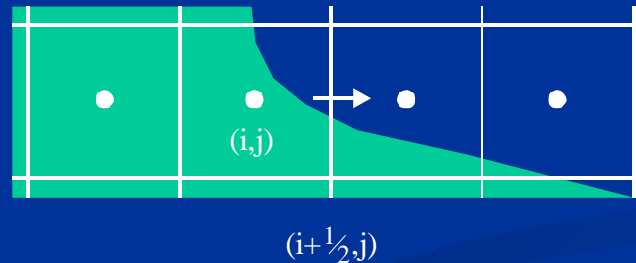
$$\varepsilon = \frac{U_*^3}{y\kappa}$$

Problem praćenja slobodnog nivoa vode:

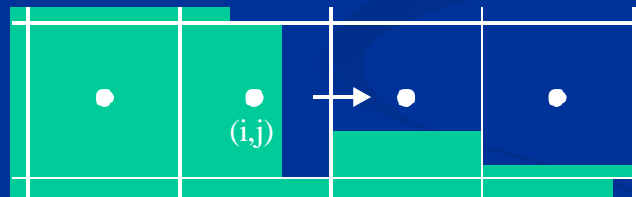
- MAC (Welch et al., 1965.)
- Density function method (Asai & Tsubogo, 2005)
- VOF metod (Hirt & Nichols, 1981.):

$$\frac{\partial F}{\partial t} + \frac{\partial}{\partial x_i} [FU_i] = 0$$

F – cell saturation



Real free surface



Approximation

$$q_{i+1/2,j} = \text{MIN} \left\{ F_{i+1,j} U_{i+1/2,j} \delta t + \text{MAX} \left[0.0, (1.0 - F_{i+1,j}) U_{i+1/2,j} \delta t - (1.0 - F_{i,j}) \delta x \right], F_{i,j} \delta x \right\}$$

Ukoliko se "nema vremena" za programiranje, postoje i besplatna rešenja

- **iRIC software** : <http://i-ric.org/en/>

